

FOR MARINE TECHNOLOGY AND INDUSTRIAL AERODYNAMICS

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Southampton Marine and Maritime Institute

Evidencing a Seastate Allowance for Inshore Fishing Vessels by Model Scale Testing Phase I

EXECUTIVE SUMMARY

This report describes a programme of model scale tests of a 10 metre fishing vessel monohull design. The test vessel, a double chine steel potter of typical proportions, was tested at two configurations – decked vessel and open boat – over a range of loading conditions representing fishing operations. The tests were conducted in the 138 metre Boldrewood Towing Tank at the University of Southampton.

This work supports an ongoing project of the National Federation of Fishermen's Organisations (NFFO) and the objectives are twofold:

- establish whether the decked vessel configuration is sufficiently safe from capsizing in waves when operated within a seastate orientated allowance, as defined by the Wolfson Stability Method.
- determine the survivability in waves and, therefore, the relative safety from capsizing of the vessel at two configurations: decked and open.

The test vessel was selected in discussion with the Marine Accident Investigation Branch (MAIB) and the data available to the Wolfson Unit included a 3d definition of the hull and the vessel's stability history.

A 1:6 scale physical model was built and load conditions were identified whereby the decked vessel would fail a statutory stability assessment but, according to the Wolfson Method, could still be operated safely in benign seastates, Physical testing at these conditions confirms the merit of a seastate-based allowance, since all capsize events occurred at combinations of stability and wave height deemed unsafe by the Wolfson Method.

The decked model was then re-configured as an open boat by blocking up its freeing ports, installing a deck drainage system designed to the current MCA Code of Practice (CoP) and fitting a bilge pump of the appropriate capacity. The open boat model was then tested in the same conditions as the decked vessel.

Both the decked and the open vessel configuration capsized in short, steep regular waves approximately 1.1-1.2m in height. In a near-accident condition, the decked model also capsized within the wave height range 1.3 -3.0m whereas the open model survived.

At the MCA minimum rating of 190 litres per minute, the bilge pump appeared adequate to disperse the excess flood water and therefore prevent a capsize. However, in a scenario with a malfunctioning bilge pump, the number of capsize events could potentially rise from 2 to 8 out of the 31 open boat runs performed.

It is recommended to evaluate the merit of fitting non-return flaps to the freeing ports of existing low-freeboard fishing boats, to consider replacing the CoP distance-based restrictions with seastate-based guidance according to the Wolfson Method, and to permit decked monohulls similar to the test vessel to demonstrate CoP compliance via alternative routes, as opposed to encourage their conversion to open boats.



1 INTRODUCTION

This report describes Phase I of a study to provide the NFFO with experimental evidence on the survivability in waves of a typical 10 metre fishing vessel at two configurations, decked vessel and open boat, in relation to stability criteria and guidance presented in the current MCA Code of Practice. The HEIF-RCSF grant funding that underpins this work was awarded by the Southampton Marine and Maritime Institute (SMMI) on 20th January, 2023 following proposal ref. 5358ms-wd Rev2 of 18th November, 2022.

This study was conducted by the Wolfson Unit in partnership with NFFO and Public Policy | Southampton (PPS), with the MCA acting as independent observers.

2 BACKGROUND

Commercial fishing remains the most dangerous peacetime occupation worldwide. The death rate in developed countries exceeds 100 fatalities per 100,000 active fishers and is likely to be even higher in developing countries, where no formal accident reporting systems are in place [2].

Small fishing vessels undertake low-impact, environmentally sustainable fishing activities, catching high quality fresh fish daily, often with the owner and family members as fishing crews [3]. These boats, however, are the most vulnerable to accidents at sea due to their size relative to the sea state and are the most likely to capsize due to insufficient stability.

The UK small-scale fishing fleet consists of 4700+ boats under 15m that is, 90% of the commercial fishing fleet.

3 RATIONALE

The new Code of Practice (CoP) for the Safety of Small Fishing Vessels of less than 15m Length Overall [4], that came into effect on 6/9/2021, is an unprecedented effort to improve safety within the small-scale fishing industry and aims at reducing the rate of fatal fishing incidents in the UK. Whilst the CoP applies to the entire under 15m fishing fleet, vessels that pre-date the new standard were not designed with it in mind and, therefore, may not fully align with the new requirements.

The CoP has a two year phase-in period ending 6/9/2023, but there are no transitional arrangements for stability. Vessels which fail to comply with the new stability requirements may not operate commercially until they demonstrate compliance, or equivalence with recognised stability criteria.

Paragraph 3.12.4 of the new CoP applies to all *existing* vessels (ie registered for the first time as a Fishing Vessel before 16/7/2007) and states:

3.12.4 Decked Vessels with freeboard less than 300 mm are to be limited in their area of operation to 20 miles from a safe haven and in favourable weather conditions. The minimum freeboard should be at least 200 mm. Vessels with less than 200 mm Freeboard are to be considered Open Vessels³.

Paragraph 3.12.4 above refers to footnote 3, which states:

 $(^{3})$ Where the freeboard is less than the minimum freeboard indicated, where equivalence can be demonstrated the MCA may accept alternative arrangements.

Paragraph 11.1 of Marine Guidance Note (MGN) 503F Amt.1 [5] enables conditional certification, ie:

11.1 As a result of the stability tests, assessment of freeboard and / or water-freeing arrangements, the attending surveyor will take into consideration the Wolfson Stability Notice and may propose that

conditions be placed on the fishing vessel certificate. The intention of these conditions is to allow the vessel to operate in a controlled manner within a geographical / distance from safe haven / weather orientated allowance.

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If equivalence cannot be demonstrated and conditional certification is not offered, then low freeboard decked vessels may either cease commercial operation or be converted to open vessels as stated in MCA guidance [6] below:

6. Freeboard (..) Vessels with less than 200mm freeboard may submit proposals to be considered for equivalence with vessels with 200mm freeboard or over. Vessels which do not achieve equivalence will be considered open vessels and the owner will need to bring the vessel in line with the open vessel standards, including water freeing arrangements.

Decked vessels may be converted to open boats by blocking up the freeing ports and installing alternative arrangements for dispersing flood water, such as electric and/or manually operated bilge pumps, depending on the vessel's size. In particular, the CoP open vessel requirements include:

2.19.4.1 There should be effective drain openings fitted on each side of the sole deck to enable any water to drain directly to the bottom of the Vessel. In the case of a Vessel with a sealed sole deck, an aft sump is to be fitted, extending from the keel to deck; potters and creel boats may have a sump located adjacent to the hauler position.

2.19.4.2 The drainage area should be at least 2% of the total bulwark area above the sole;

2.19.4.3 Open Vessels are not to be fitted with freeing ports;

The level of safety associated with the seastate based certification and the conversion of *existing* decked vessels to open boats are ongoing concerns for the inshore fishing industry and require evidencing.

4 **PROGRAMME OF WORK**

4.1 Work package 1 – selection of a test vessel

Several low-freeboard, decked fishing vessel designs under 15m overall length were shortlisted for the tests and a final design was selected in discussion with the Marine Accident Investigation Branch (MAIB). The data available for the test vessel include: a table of offsets of the hull to the top of the gunwale, general arrangement, tank definitions and capacity tables, inclining experiment results, two lightship conditions (as-built and modified, following the vessel's re-purposing) and a comprehensive stability assessment over a range of loading conditions representing typical fishing operations.

Table 1 presents the vessel's principal dimensions, alongside the datums and coordinate system used in this report.

4.2 Work package 2 – digital stability modelling of the decked vessel configuration

The WP1 data were processed to produce a digital stability model in the Wolfson hydrostatics and stability suite (HST). Subsequently, the model was successfully validated against the original hydrostatics following the Wolfson Unit's standard procedure and quality assurance criteria. The validated hull definition enabled a full stability analysis including assessment against the appropriate stability requirements, in accordance with [4] and [5].

Figures 1 and 2 present the HST hull definition.



The WP2 results are given in Section 5 and discussed in Section 6.

4.3 Work package 3 – survivability tests at model scale, decked vessel configuration

A 1:6 scale model of the vessel was constructed of wood and GRP in accordance with the WP2 data. The model was designated as Wolfson Unit model M1203 and incorporated a cambered main deck and a cambered foredeck, both fitted with hatches for ballasting purposes, a ballasting rig above deck to enable large changes in the vessel's centre of gravity position and a fixed bulwark fitted with rectangular freeing ports.

The survivability tests in waves were conducted in the Boldrewood Towing Tank, University of Southampton. The tank is 145m long, by 6m wide, by 3.5m deep and is fitted with a wavemaker capable of producing full scale regular waves up to 4.2 metres high at a full scale period of 5.8 seconds.

Table 2 describes the water freeing arrangements of the decked vessel, which are at the minimum MCA requirement. Figure 3 shows the decked vessel in the Boldrewood Tank.

The WP3 test technique and results are given in Section 7 and discussed in Section 8.

4.4 Work package 4 – survivability tests at model scale, open boat configuration

The WP3 decked vessel model was converted to an 'equivalent' open boat by blocking up the freeing ports and fitting a bespoke, 3d printed drainage arrangement. Such an arrangement, typically referred to as 'sump and pump', consisted of a deck opening on each side of the vessel's centreline at the lowest point of the sheer, leading to a centreline containment tank fitted with a remotely controlled bilge pump.

The CoP bilge pumping requirements for *existing* open vessels are:

4.10.4 (Existing Vessels) Where standards do not exist the bilge pumping arrangements must be fit for purpose.

It is understood from the MCA that Marine Guidance Note 628 (M+F) 'Construction and outfit standards for fishing vessels of less than 15m length overall' Part 9 'Pumping and piping systems' is typically used for assessing the fitness for purpose of the bilge pumping arrangements of *existing* open vessels not built to a construction standard. In particular, paragraph 9.3.2 of MGN 628 requires vessels of 10 to 15m to install two bilge pumps (one manual, one electrical) having a minimum combined capacity of 190 litres/minute.

The bilge pump installed in the model was calibrated in-situ over a range of static heads and was operated at the minimum MCA requirement of 190 litres/minute throughout the tests.

Table 3 describes the water freeing arrangements of the open boat configuration. Figures 4 and 5 show the model in the open boat configuration and the bilge pump installation, with the main deck hatch temporarily removed.

The WP4 test technique and results are given in Section 9 and discussed in Section 10.

5 WP2 DIGITAL STABILITY MODELLING OF DECKED VESSEL: RESULTS

The resulting data are shown in the following tables:-

- Table 4Decked Vessel Stability Notice and Freeboard Guidance Mark
- Table 5Undecked Vessel Stability Notice and Freeboard Guidance Mark

Table 6	Loading Conditions and Configurations
Table 7	Summary of Stability and Maximum KG Criteria

The full stability results are presented in Appendix 1.

6 WP2 DIGITAL STABILITY MODEL OF DECKED VESSEL: DISCUSSION

a. Loading conditions TEST 0 to 4 of Table 6 were assessed against the CoP criteria of para. 3.5. Pre-2007, unmodified vessels are not required to carry approved stability information and therefore need not meet such criteria. However, pre-2007 vessels that carry out modifications, alterations or change fishing method are treated as new vessels (CoP para 3.12.7) and may choose to be assessed against the CoP criteria in order to demonstrate suitable stability characteristics.

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- b. TEST 0, the as-built port departure condition, met all the CoP stability criteria and exceeded the minimum freeboard requirement by more than 30%. Subsequent modifications eroded the vessel's stability reserves, resulting in partial compliance at TEST 1-2 and no compliance at TEST 3-4. TEST 3 is a top-heavy operating condition and TEST 4 is a near-accident condition.
- c. The Wolfson Stability Method indicates that the test vessel in the decked configuration can safely operate in the amber zone at port departure (TEST 0, 1) and at port arrival (TEST 2.1 and 3.1), but is at risk of capsize in the overloaded port arrival condition TEST 4.1. For the test vessel, the maximum recommended Hs in the amber zone is 1.3m that is, low end of Douglas 4 'Moderate'.
- d. TEST 0, 1, 2.1 and 3.1, however, are upright conditions. Lifts over the side, transverse shifts in catch/ gear or gear snags resulting in 5° heel or more would reduce the residual freeboard to an unsafe level and the Wolfson method predicts that the vessel would be in danger of capsizing in sea states exceeding Hs = 0.6m that is, low end of Douglas 3 'Slight' at the heeled conditions tested.

7 WP3 DECKED VESSEL SURVIVABILITY: TEST TECHNIQUE AND RESULTS

The model was ballasted to the TEST 1 condition, at 13.54t displacement, -4.814m LCG and an inclining in air was performed to adjust its VCG to 1.391m. The pitch gyradius was set to 25% overall length and the roll gyradius to 33% overall beam. Colour-coded vertical bands were painted on each side of the model at two locations (25% overall length and midships) to identify the Wolfson Guidance Mark and associated safety zones.

The model was tested over a range of loading conditions, wave heights and periods following the test protocol described in [7]. In preparation to each run, the onboard ballast was adjusted to achieve the desired loading condition and direct measurements were made of the static heel and trim angles using a digital inclinometer with 0.01 degree accuracy, to ensure consistency with the calculated heel and trim of Table 7. The model was then positioned on the towing tank centreline and approximately 20 metres from the wavemaker.

The model heading was controlled manually with light lines fore and aft to enable a continuous rotation over the appropriate range of headings within each run. Each run comprised a continuous 180° rotation from 0° heading ie bow towards the incoming waves to 180° heading ie stern towards the waves. In addition, if the model had an initial list, it was tested at additional headings with the low side towards and away from the incoming waves to ascertain the most vulnerable side.

All the physical tests presented in this report were conducted in regular waves. This approach enables to correlate the minimum wave height to capsize the model (as measured in the towing tank with a wave probe) with the full scale sea state where it is probable to encounter such a 'critical' wave at least once during a typical

inshore fishing trip. For example, formulae given in [8] indicate that approximately 1 in 2000 waves within a typical seastate (Rayleigh distributed spectrum) will be twice the significant height. The reader is referred to [7] for a full discussion on this subject.

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The resulting data are shown in the following figures:-

- Figure 6 Regular waves measured at loading conditions tested
- Figure 7 Capsize boundary for configuration ID 2.x, full scale data
- Figure 8 Capsize boundary for configuration ID 4.x, full scale data
- Figure 9 Relationship between the condition tested and the Wolfson safety zones
- Figure 10 Comparison of predicted critical seastates and actual capsize wave heights
- Figure 11 Variation of Minimum Wave Height to Capsize / LOA with Stability

8 WP3 DECKED VESSEL SURVIVABILITY: DISCUSSION

- a. The 6 load conditions tested resulted in 3 being capsized and 3 surviving. 7 capsize events in total were observed and the capsize mechanism appeared to be consistent across all these events.
- b. The 3 capsized conditions exhibited an initial list of 5° or more. The model consistently capsized whilst beam on and listing towards the incoming waves ie 'to windward'. The capsize mechanism entailed rolling of the model to large windward angles, causing immersion of the windward freeing ports and resulting in the accumulation of water on deck. The net amount of water on deck increased over time, either progressively through the freeing ports or suddenly over the gunwale, until the model capsized to windward.
- c. 5 out of 7 capsizes occurred at condition 4.2 where a systematic variation of wave height and period was performed within the wave envelope permitted by the wave maker. The capsize boundary for condition 4.2 is given in Figure 8b and indicates that steeper waves increase the risk of capsizing.
- d. When the test vessel has less than 210mm residual freeboard, the Wolfson Guidance recommends a maximum seastate of 0.6m significant wave height, corresponding to a maximum wave height of approximately 1.2m (1 in 2000 waves, see Section 7). The minimum wave height to capsize of Figures 7c and 8b are within 13% of this prediction, which is in keeping with the general level of approximation provided the Guidance.
- e. Figure 8c shows that, on one occasion, the model capsized below the maximum recommended wave height based on the residual freeboard, but well above the predicted critical wave height based on the residual stability. It is worth noting that this capsize occurred at a near-accident condition (TEST 4.3) where the vessel was overloaded and had 9° of list, resulting in a minimum freeboard of minus 100mm (that is, deck edge underwater), a positive range of stability of just 7 degrees and a maximum righting moment of only 90 kg.m. At this loading condition, the stability reserves are small and the vessel may easily capsize in calm conditions due to small transverse weight shifts (eg crew, gear or catch).
- f. The type of plot shown in Figure 11 was first presented in [7] and has subsequently been updated to include additional vessel casualties up to 2021 [9, 10]. The solid diagonal line represents the minimum wave height to capsize a vessel with known residual stability characteristics, according to the Wolfson Method. Combinations of stability and seastate below the line represent operating conditions where the vessel is deemed safe from capsizing, whereas the zone above the line indicates danger of capsize. Figure 11 was used to summarize the outcome of the survivability tests conducted on the decked vessel and presented in this report. All the combinations of stability (calculated) and seastate (measured) resulting in a capsize lie on or above the line, which confirms the validity of the Wolfson Method for the hull form tested. These results are in keeping with other 'stress testing' exercises conducted by the Wolfson Unit, such as that described in [10] and summarised in Figure 20.



9 WP4 OPEN BOAT SURVIVABILITY: TEST TECHNIQUE AND RESULTS

The decked vessel test programme of Section 7 was repeated with the model in the open boat configuration and the same test technique was used. The tests were conducted at a constant bilge pump capacity of 190 litres per minute, which is the minimum MCA requirement stated in MGN 628(M+F), as discussed in Section 4.4.

At the near-accident condition 4.3, the decked vessel at equilibrium in calm water had water on deck due to the immersion of its freeing ports. When ballasted to condition 4.3, the open boat model had a dry deck as the freeing ports were blocked up, so it exhibited no free surface effects and no VCG increase due to the water on deck, resulting in better overall stability. Therefore, no survivability tests were conducted on the open boat model at condition 4.3, as the different initial stability would prevent a direct comparison with the decked vessel results.

The resulting data are shown in the following figures:-

Figure 6Regular waves measured at loading conditions testedFigure 7a to cCapsize boundary for configuration ID 2.x, full scale dataFigure 8a to cCapsize boundary for configuration ID 4.x, full scale data

Finally, a simple numerical study was conducted in two water draining modes and the resulting GZ curves are shown in the following figures:

Figure 12	Open boat variant: Righting lever (GZ) curves in 'sump & pump' mode
Figure 13	Open boat variant: Righting lever (GZ) curves in 'free draining to bilge' mode
Figure 14	Open boat variant: Righting lever (GZ) curves with 1.15 t flood water
Figure 15	Open boat variant: Righting lever (GZ) curves with 1.65 t flood water
Figure 16	Open boat variant: Righting lever (GZ) curves with 2.15 t flood water
Figure 17	Open boat variant: Righting lever (GZ) curves with 2.65 t flood water

10 WP4 OPEN BOAT SURVIVABILITY: DISCUSSION

- a. The 5 load conditions tested resulted in 2 being capsized (nos. 2.3 and 4.1, same as the decked vessel) and 3 surviving. 2 capsize events in total were achieved and the capsize mechanism appeared to be consistent across these events. The 2 capsized conditions exhibited an initial list of more than 5°.
- b. Similarly to the decked vessel, the open boat capsized in short, steep regular waves approximately 1.1-1.2 metres in height. However, the capsize mechanism of the open boat was different, as it was initiated by a single wave hitting the side of the vessel while upright and beam on to the waves, Figures 18 19. Such a wave deposited a large amount of water on deck, which overwhelmed the bilge pump resulting in increased windward heel, reduced freeboard and therefore greater vulnerability to swamping by subsequent waves.
- c. When tested at load condition 2.3, both the decked vessel and the open boat capsized at approximately the same combination of wave height and period, 1.15 metres and 2.75 seconds respectively.
- d. When tested at load condition 4.2, both the decked vessel and the open boat capsized at exactly the same combination of wave height and period, 1.20 metres and 2.80 seconds respectively. The decked vessel also capsized within the wave height range 1.3 3.0 metres whereas the open boat survived within, and at selected wave periods beyond, that range. This can be explained by the different capsize mechanisms of the decked vessel (item 8.b) and the open boat (item 10.b).
- e. In short, steep waves the open boat often exhibited a wet deck, whereby moderate amounts of water landed on it at every wave encounter and at most headings. At the MCA minimum rating of 190 litres per minute, the bilge pump capacity was adequate to disperse the excess flood water and therefore prevent a capsize. If the bilge pump malfunctioned in these 'marginal' conditions, it is highly likely that the vessel would have capsized, thus increasing the number of capsize events from 2 to 8 out of 31 runs performed in total.
- f. In long, steep waves and stern quartering seas, the aft bulwark was often overwhelmed by the incoming waves. Had the model been fitted with a shooting opening as per the actual vessel design, the vessel would have been vulnerable to swamping in following seas.
- g. Whilst the model configuration tested is deemed 'open' from a regulatory perspective (as it does not have a *continuous watertight weather deck with positive freeboard in all loading conditions*, [11]) and meets the MCA water freeing requirement for *existing* open vessels, it deviates substantially from conventional open boat designs. For this reason, the open boat described in this report was not assessed against the Wolfson Stability Notice and Freeboard Guidance Mark which were derived from, and are therefore only applicable to, conventional open boats.
- h. The total capacity of the drainage system fitted to the model tested (Figure 4) is 650 litres at full scale. If the test open boat had a bilge pumping failure, up to 650 litres of flood water would accumulate in the draining system and, therefore, act as water ballast below the deck. Flood water in excess of 650 litres would accumulate above deck and the stability would rapidly deteriorate due to increasingly large free surface effects, resulting in a gradually rising VCG. This scenario is described in Figure 12, showing the vessel's righting lever curve with full drains (0.65t flood water) and, in addition, variable amounts of water on deck, from 0.5t to 2.0t. The analysis shows that, as water accumulates on deck, the vessel develops an increasing angle of loll (ship unstable when upright).
- i. If the deck drained freely to the bilge rather than to a containment tank and a bilge pumping failure occurred, larger amounts of flood water could accumulate below the deck than in the 'sump and pump' case, hence the VCG rise due to the flood water would be less. This is demonstrated by Figures 14 to 17 where the 'free draining' GZ curve dominates the 'sump & pump' GZ curve at any given level of flood water over the range 1.15 2.65 t.

11.1 The decked vessel capsize mechanism consistently entailed the progressive accumulation of water on deck through the freeing ports. Therefore, further model scale testing is recommended to evaluate the merit of retrofitting non-return flaps to the freeing ports of existing decked fishing vessels with less than 300mm freeboard to improve their survivability in waves.

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- 11.2 For the decked vessel tested, short and steep waves as may be encountered in coastal sea states proved just as critical as bigger waves representing further offshore conditions. This challenges the distance from safe haven restrictions imposed by the new Code of Practice on 200 to 300mm freeboard vessels. It is therefore recommended to either support such distance-based restrictions with further evidence or consider their replacement with seastate-based guidance according to the Wolfson Stability Method.
- 11.3 At the combinations of regular waves and load condition tested, the decked vessel configuration appeared just as vulnerable to capsizing as the 'equivalent' open vessel in short, steep waves representing coastal seastates where inshore fishing boats would typically operate. Therefore, it is recommended that existing decked monohulls whose dimensions and stability characteristics are similar¹ to the 10m vessel tested are not converted to open vessels in order to meet the Code of Practice water freeing requirements. Such vessels should be permitted to demonstrate compliance via an alternative route, for example a residual freeboard assessment in the worst foreseeable operating condition.

¹ with respect to dimensional similarity, freeing port arrangement and residual stability characteristics.



12 REFERENCES

- [1] Maritime and Coastguard Agency. Marine Guidance Note 526F 'Stability guidance for fishing vessels - Wolfson method', November 2018.
- [2] Willis et al. 'The human cost of global fishing', Marine Policy 148, 2023.
- [3] Low Impact Fishers of Europe, https://lifeplatform.eu
- [4] Maritime and Coastguard Agency. 'The Code of Practice for the Safety of Small Fishing Vessels of less than 15m Length Overall', last updated 19th August 2022.
- [5] Maritime and Coastguard Agency. Marine Guidance Note 503 Amt. No.1 (F) 'Procedure for Carrying out Small Fishing Vessel Stability Tests', September 2022.
- [6] Maritime and Coastguard Agency. 'Response to queries on the revised code of practice for the safety of small fishing vessels of less than 15m length overall (2021)', (https://www.gov.uk/government/publications/applying-the-code-of-practice-for-the-safety-of-small-fishing-vessels-of-less-than-15m-length-overall/response-to-queries-on-the-revised-code-of-practice-for-the-safety-of-small-fishing-vessels-of-less-than-15m-length-overall-2021--2), 5 May 2021, date of last access: 10 July 2023.
- [7] Wolfson Unit. 'MCA Research Project 509: High Speed Craft Evaluation of Existing Criteria, Final Report'. Report No.1779, Maritime & Coastguard Agency, UK.
- [8] Price, W.G. and Bishop, R.E.D., 'Probabilistic Theory of Ship Dynamics', Chapman & Hall Ltd., 1974.
- [9] Deakin, B., 'Spend Less, Save More (Lives)', Proc. of the 11th International Conference on the Stability of Ships and Ocean Vehicles (STAB 2012), Athens, Greece, 2012.
- [10] Scarponi, M., 'Stress Tests on the Wolfson Stability Method for Small Fishing Vessels'. Wolfson Unit Report No. 2857, Lloyd's Register Foundation Small Grant Final Report Sg4\100045, 2022.
- [11] Maritime and Coastguard Agency. Annex 8 'Definitions of Decked and Open Vessels' to Merchant Shipping Note 1871 Amt. 2 'The Code of Practice for the Safety of Small Fishing Vessels of less than 15m Length Overall', last updated August 2022.



Table 2 Water Freeing Arrangements as Decked Vessel

Length / Beam Ratio	2.98
CoP requirement for <i>existing</i> vessels of L/B > 2.5	Para 2.20.5: 4% tot. bulwark area each side
Total bulwark area each side (m ²)	5.266
Required freeing port area, each side (m ²)	0.211
Actual freeing port area, each side (m ²)	0.211

All data at full scale unless otherwise noted

Figure 3 shows a bow quarter view of the decked vessel configuration.

CoP requirement for <i>existing</i> open vessels	Para 2.19.4.2 The drainage area should be at least 2% of the total bulwark area above the sole;
Total bulwark area each side (m²)	5.266
Required drainage area, each side (m ²)	No less than 0.105
Actual drainage area, each side (m²)	0.105
Long edge of drainage opening (m)	0.420
Short edge of drainage opening (m)	0.251
Capacity of centreline containment tank (litres)	319
Capacity of each duct (litres)	159
Bilge pump capacity (litres / minute)	190

Table 3Water Freeing Arrangements as Open Vessel

All data at full scale unless otherwise noted

Figure 4 shows the drainage arrangement installed in the open boat configuration.

STABILITY NOTICE Name 0 No. M1203 Loading Safety Minimum Maximum Owner 0 & Lifting Freeboard Recommended Zone Length 10.43 metres Guidance Seastate Beam 3.498 metres Good Good margin of margin of At least 43 cm residual freeboard safety Loading or lifting reduces minimum Low level 1.3 metres 21 to 43 cm freeboard to less of safety than 43 cm Excessive loading Danger of or lifting reduces Less than 21 0.6 metres minimum freeboard capsize cm to less than 21 cm

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Table 4 Decked Vessel Stability Notice and Freeboard Guidance Mark



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Table 5

Table 6Loading Conditions and Configurations

ID	Parent Lightship	Description	Displ.	LCG	VCG fluid	Configurations Calculated	Notes
			tonnes	metres	metres	metres	
LIGHT 0	n/a	As-built (1987) Lightship	10.75	-4.497	1.140	n/a	- unmodified, steel-hulled potter design
LIGHT 1	n/a	Modified (2021) Lightship	11.59	-4.695	1.295	n/a	- re-purposed, heavy gear added above deck - re-ballasted
TEST 0	LIGHT 0	98% Departure Port	12.70	-4.654	1.267	Upright	- as-built departure - passes all CoP New Vessel criteria - fails MGN 503F Offset Load Test - min. upright freeboard: Amber
TEST 1	LIGHT 1	98% Departure Port	13.54	-4.814	1.391	Upright	- modified departure - passes some CoP New Vessel criteria - fails MGN 503F Offset Load Test - min. upright freeboard: Amber
TEST 2	LIGHT 1	10% Arrival Port, Base VCG	14.42	-4.902	1.425	Upright + 4.97° + 9.81°	- catch on deck, empty fish hold - fails most CoP New Vessel criteria - fails MGN 503F Offset Load Test - min. freeboard: Amber (upright) / Red (heeled)
TEST 3	LIGHT 1	10% Arrival Port, High VCG	14.42	-4.902	1.625	Upright + 5.81° + 11.75°	 catch on deck, empty fish hold, top heavy fails all CoP New Vessel criteria capsizes at MGN 503F Offset Load Test min. freeboard: Amber (upright) / Red (heeled)
TEST 4	LIGHT 1	Heavy 10% Arrival Port, High VCG	17.77	-5.220	1.616	Upright + 5.49° + 8.75°	- overloaded & top heavy - fails all CoP New Vessel criteria - capsizes at MGN 503F Offset Load Test - min. freeboard: Red

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Table 7Summary of Stability and Max KG Criteria

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				see: \4200 Seastate allowance Phase 1 (HEIF)\WP2 Stability modelling\10m_potter_Rev14.hst										
					<u>Bul</u>	warks as p	er M1203 8	& range up t	o AVS, as r	nodel has	no downflo	oding poin	<u>ts</u>	
				TEST 0	TEST 1		TEST 2			TEST 3			TEST 4	
		Configui	ration ID =>	0	1	2.1	2.2	2.3	3.1	3.2	3.3	4.1	4.2	4.3
	Displacement	t		12.698	13.537	14.417	14.417	14.417	14.417	14.417	14.417	17.773	17.773	17.773
	LCG	m from FP		-4.654	-4.814	-4.902	-4.902	-4.902	-4.902	-4.902	-4.902	-5.220	-5.220	-5.220
	VCGf	m abv base		1.267	1.391	1.425	1.425	1.425	1.625	1.625	1.625	1.616	1.616	1.616
	TCG	m		0.000	0.000	0.000	0.044	0.090	0.000	0.033	0.066	0.000	0.027	0.040
_	Draught @ AP	m abv base		1.040	1.137	1.201			1.201			1.467		
Data	Draught @ FP	m abv base		1.123	1.086	1.086			1.086			1.052		
n [Equilibrium Heel Angle	degrees (+ve stb	od down)	0.00	0.00	0.00	4.80	9.83	0.00	5.83	11.79	0.00	5.52	8.96
ditio	Freeboard min.	m from deck edg	ge	0.399	0.365	0.322	0.190	0.052	0.322	0.162	-0.001	0.120	-0.020	-0.112
ono	Freeboard min. location	% LOA		54%	44%	35%	44%	44%	35%	44%	44%	9%	17%	17%
0	Pos. Clear Height at Side	m from gunwale		1.073	1.043	1.009	0.866	0.712	1.009	0.835	0.653	0.807	0.662	0.559
	Wolfson Safety Zone	as decked		Am	Am	Am	Rd	Rd	Am	Rd	Rd	Rd	Rd	Rd
	Wollson Salety Zolle	as undecked		Am	Am	Am	Am	Am	Am	Am	Am	Am	Am	Am
	Trim over LBP	m (-ve by bow)		0.724	0.721	0.115			0.115	0.000	0.000	0.415	0.000	0.000
	Trim angle	deg (-ve by bow)	4.29	4.27	0.68			0.68			2.46		
	Downflooding angle	deg to stbd		99.0	36.2	34.5	34.5	34.5	34.8	34.5	34.5	29.3	29.3	29.3
Data	Max GZ	m		0.247	0.181	0.157	0.117	0.075	0.085	0.053	0.022	0.044	0.018	0.005
1 ZE	Positive Range (ignore df)	deg		86.0	63.6	56.8	48.2	38.1	37.7	28.4	16.5	26.5	15.4	7.8
Ŭ	GMs	m		0.674	0.564	0.529			0.329			0.312		
	1: GZ area to 30°	m.rad	0.055	0.08	0.064	0.057	0.037	0.020	0.030	0.015	0.004	0.013	0.003	0.000
erië	2: GZ area to 40° or df	m.rad	0.09	0.123	0.093	0.080	0.054	0.030	0.034	0.016	0.004	0.013	0.003	0.000
Crit	3: GZ area 30° to 40° or df	m.rad	0.03	0.043	0.029	0.023	0.017	0.010	0.004	0.001	0.000	0.000	0.000	0.000
lity	4: Max GZ Angle	deg	25	34.2	25.8	23.9	24.3	25.4	18.7	19.3	19.6	12.3	12.5	12.5
abi	5: GZmax abv 30°	m	0.2	0.247	0.178	0.150	0.112	0.072	0.050	0.022	-0.007	-0.023	-0.046	-0.057
St	6: GMf	m	0.35	0.672	0.563	0.526	0.522	0.513	0.326	0.321	0.302	0.275	0.285	0.149
c	Hcrit known stability	m full scale		4.354	2.846	2.443	1.790	1.132	1.193	0.709	0.266	0.670	0.249	0.067
fsoi	nent, known stabinty	m 1:6 scale		0.726	0.474	0.407	0.298	0.189	0.199	0.118	0.044	0.112	0.041	0.011
Vol	Herit unknown stability	m full scale		2.548	2.548	2.548	1.274	1.274	2.548	1.274	1.274	1.274	1.274	1.274
>		m 1:6 scale		0.425	0.425	0.425	0.212	0.212	0.425	0.212	0.212	0.212	0.212	0.212
3F ad	Equilibrium heel (Move 2)	deg	15	6.9	7.8	7.8								
- 150	Res. Freeboard (Move 2)	mm	75	178	124	83								
1GN fset	Equilibrium heel (Move 3)	deg	15	10.0	11.3	11.4								
≥ ₽	Res. Freeboard (Move 3)	mm	75	81	15	-24								





Figure 1Bow Quarter View of Digital Hull Definition







Figure 3 Decked vessel variant under test, 17.77 tonnes displacement, load condition 'TEST 4.2'







Figure 4 Drainage Arrangement for Open Boat Variant: Detail of Sump and Drain Openings







Figure 5 Drainage Arrangement for Open Boat Variant: Details of Bilge Pump Installation

Figure 6 Regular waves measured at loading conditions tested



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 Figure 7b
 Tests performed for configuration ID 2.2, full scale data

 Max. Recommended Regular Wave Height* for decked configuration at < 21cm freeboard = 1.27m</td>

 ID 2.2 (decked) freeboard ~19 cm







(*) as predicted by the Wolfson Method











(*) as predicted by the Wolfson Method



Figure 9 Decked vessel: relationship between the condition tested and the Wolfson safety zones





Figure 11 Decked vessel: variation of minimum Wave Height to Capsize / LOA with Stability

Note: each error bar defines the minimum and maximum measured wave height up to the capsize event.





Figure 12 Open boat variant: Righting Lever (GZ) curves in 'sump & pump' mode

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Angle - degrees



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Figure 18 Open boat, loading condition 2.3, run 49: wave height 1.18 metres, wave period 3.0 seconds



Figure 19 Open boat, loading condition 4.2, run 59: wave height 1.20 metres, wave period 2.8 seconds





Figure 20 Correlation of Casualty and Model Data with the Wolfson Formula, Ref. [10]



Note: most accident investigation reports present the probable sea state at the time of the accident as a value on the Douglas Scale. This corresponds to a range of significant wave heights:

eg Hs = 0.5 to 1.25m for Douglas 3 'Slight'.

For these F/V casualties, the wave height plotted in Figure 20 corresponds to the minimum of such a Hs range:

eg H = $2 \times 0.5 = 1$ m (assumes 1 in 2000 waves will be twice the significant height, see Section 7 above)

which brings these data points closer to the Wolfson line, resulting in a more onerous stress test.



Appendix 1

Intact Stability Data



Filename: Z:\Projects\4200 Seastate allowance Phase 1 (HEIF)\WP2 Stability modelling\10m_potter_Rev12.hst Date : 05/06/2023 Time :08:43:07

Hull

Ship Particulars

Roll Centre	2.000 metres
Specific Gravity of Water	1.0250
Mean Shell Thickness	0.0000 metres
Longitudinal Datum	Fore Perpendicular
Vertical Datum	USK Amidships (X=4.825m)
Trim Length	9.650 metres

Draught Marks Name X metres Z metres

Aft Marks	A.P.	-9.650	0.000
Mid Marks	Midships	-4.825	0.000
Fwd Marks	F.P.	0.000	0.000

Conditions

1: TEST 0 Disp: 12.698 tonnes, LCG: -4.654 metres, VCG: 1.267 metres

2: TEST 1 Disp: 13.537 tonnes, LCG: -4.814 metres, VCG: 1.391 metres

3: TEST 2.1 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.425 metres 4: TEST 2.2 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.425 metres 5: TEST 2.3 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.425 metres

6: TEST 3.1 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.625 metres 7: TEST 3.2 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.625 metres 8: TEST 3.3 Disp: 14.417 tonnes, LCG: -4.902 metres, VCG: 1.625 metres

9: TEST 4.1 Disp: 17.773 tonnes, LCG: -5.220 metres, VCG: 1.616 metres 10: TEST 4.2 Disp: 17.773 tonnes, LCG: -5.220 metres, VCG: 1.616 metres 11: TEST 4.3 Disp: 17.773 tonnes, LCG: -5.220 metres, VCG: 1.616 metres

TEST 0

Displacement	12.698 tonnes
Longitudinal Centre of Gravity	-4.654 metres
Vertical Centre of Gravity	1.267 metres
Transverse Centre of Gravity	0.000 metres
Equilibrium GM	0.672 metres
Equilibrium Heel Angle	0.000 degrees
Equilibrium Draught	1.081 metres
Equilibrium Trim Between Marks	0.083 metres by the bow
Angle of Vanishing Stability	86.0 degrees
Maximum GZ	0.247 metres
Maximum GZ Angle	34.2 degrees

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	0.000	0.000	1.123	-0.083	0.737	0.000
5.0	0.059	0.169	1.128	-0.090	0.742	0.003
6.2	0.072	0.209	1.130	-0.094	0.744	0.004
10.0	0.116	0.336	1.141	-0.110	0.755	0.010
12.2	0.140	0.408	1.148	-0.123	0.764	0.015
15.0	0.170	0.498	1.159	-0.140	0.777	0.023
20.0	0.214	0.647	1.185	-0.177	0.805	0.040
25.0	0.237	0.772	1.218	-0.218	0.833	0.059
30.0	0.245	0.879	1.256	-0.253	0.861	0.080
35.0	0.248	0.974	1.292	-0.278	0.892	0.102
40.0	0.239	1.054	1.330	-0.293	0.921	0.123
45.0	0.224	1.120	1.366	-0.290	0.950	0.144
50.0	0.205	1.175	1.399	-0.268	0.979	0.162
55.0	0.180	1.218	1.432	-0.234	1.008	0.179
60.0	0.153	1.250	1.469	-0.195	1.038	0.194





TEST 1

Displacement	13.537 tonnes
Longitudinal Centre of Gravity	-4.814 metres
Vertical Centre of Gravity	1.391 metres
Transverse Centre of Gravity	0.000 metres
Equilibrium GM	0.563 metres
Equilibrium Heel Angle	0.000 degrees
Equilibrium Draught	1.111 metres
Equilibrium Trim Between Marks	0.051 metres by the stern
Angle of Vanishing Stability	63.6 degrees to stbd 63.6 degrees to port
Maximum GZ	0.181 metres to stbd 0.181 metres to port
Maximum GZ Angle	25.8 degrees to stbd 25.8 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	0.000	0.000	1.086	0.051	0.759	0.000
5.0	0.049	0.170	1.091	0.044	0.764	0.002
6.2	0.061	0.211	1.093	0.041	0.766	0.003
10.0	0.097	0.338	1.103	0.026	0.777	0.009
12.2	0.117	0.411	1.110	0.016	0.786	0.013
15.0	0.141	0.501	1.120	0.002	0.799	0.019
20.0	0.171	0.647	1.145	-0.027	0.825	0.033
25.0	0.181	0.769	1.177	-0.055	0.851	0.048
30.0	0.178	0.873	1.211	-0.076	0.878	0.064
35.0	0.169	0.967	1.246	-0.088	0.906	0.079
40.0	0.148	1.042	1.284	-0.090	0.931	0.093
45.0	0.121	1.104	1.317	-0.071	0.956	0.105
50.0	0.091	1.157	1.348	-0.037	0.981	0.114
55.0	0.059	1.199	1.381	0.004	1.008	0.121
60.0	0.025	1.230	1.417	0.051	1.035	0.124





TEST 2.1

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.425 metres
Transverse Centre of Gravity	0.000 metres
Equilibrium GM	0.526 metres
Equilibrium Heel Angle	0.000 degrees
Equilibrium Draught	1.144 metres
Equilibrium Trim Between Marks	0.114 metres by the stern
Angle of Vanishing Stability	56.8 degrees to stbd 56.8 degrees to port
Maximum GZ	0.157 metres to stbd 0.157 metres to port
Maximum GZ Angle	23.9 degrees to stbd 23.9 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	0.000	0.000	1.087	0.114	0.783	0.000
5.0	0.046	0.170	1.090	0.110	0.787	0.002
6.2	0.056	0.210	1.092	0.107	0.789	0.003
10.0	0.090	0.338	1.101	0.096	0.800	0.008
12.2	0.109	0.410	1.107	0.088	0.809	0.012
15.0	0.130	0.499	1.117	0.077	0.821	0.018
20.0	0.153	0.640	1.140	0.059	0.845	0.030
25.0	0.157	0.759	1.170	0.044	0.869	0.044
30.0	0.150	0.863	1.202	0.036	0.894	0.057
35.0	0.136	0.954	1.236	0.035	0.920	0.070
40.0	0.109	1.025	1.271	0.050	0.941	0.080
45.0	0.079	1.086	1.302	0.082	0.964	0.089
50.0	0.047	1.138	1.332	0.127	0.987	0.094
55.0	0.013	1.180	1.363	0.177	1.011	0.097
60.0	-0.023	1.211	1.398	0.231	1.036	





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TEST 2.2

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.425 metres
Transverse Centre of Gravity	0.044 metres
Equilibrium GM	0.523 metres
Equilibrium Heel Angle	4.803 degrees to stbd
Equilibrium Draught	1.145 metres
Equilibrium Trim Between Marks	0.110 metres by the stern
Angle of Vanishing Stability	53.0 degrees to stbd 59.9 degrees to port
Maximum GZ	0.117 metres to stbd 0.197 metres to port
Maximum GZ Angle	24.3 degrees to stbd 23.4 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.044	0.000	1.087	0.114	0.783	0.002
4.8	0.000	0.163	1.090	0.110	0.787	0.000
5.0	0.002	0.170	1.090	0.110	0.787	0.000
6.2	0.013	0.210	1.092	0.107	0.789	0.000
10.0	0.047	0.338	1.101	0.096	0.800	0.002
12.2	0.066	0.410	1.107	0.088	0.809	0.004
15.0	0.088	0.499	1.117	0.077	0.821	0.008
20.0	0.111	0.640	1.140	0.059	0.845	0.017
25.0	0.117	0.759	1.170	0.044	0.869	0.027
30.0	0.112	0.863	1.202	0.036	0.894	0.037
35.0	0.100	0.954	1.236	0.035	0.920	0.046
40.0	0.075	1.025	1.271	0.050	0.941	0.054
45.0	0.048	1.086	1.302	0.082	0.964	0.059
50.0	0.018	1.138	1.332	0.127	0.987	0.062
55.0	-0.013	1.180	1.363	0.177	1.011	
60.0	-0.045	1.211	1.398	0.231	1.036	





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TEST 2.3

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.425 metres
Transverse Centre of Gravity	0.090 metres
Equilibrium GM	0.513 metres
Equilibrium Heel Angle	9.833 degrees to stbd
Equilibrium Draught	1.149 metres
Equilibrium Trim Between Marks	0.097 metres by the stern
Angle of Vanishing Stability	47.9 degrees to stbd 62.6 degrees to port
Maximum GZ	0.075 metres to stbd 0.240 metres to port
Maximum GZ Angle	25.4 degrees to stbd 22.6 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.090	0.000	1.087	0.114	0.783	0.008
5.0	-0.044	0.170	1.090	0.110	0.787	0.002
6.2	-0.033	0.210	1.092	0.107	0.789	0.001
9.8	0.000	0.332	1.101	0.097	0.800	0.000
10.0	0.002	0.338	1.101	0.096	0.800	0.000
12.2	0.021	0.410	1.107	0.088	0.809	0.000
15.0	0.043	0.499	1.117	0.077	0.821	0.002
20.0	0.068	0.640	1.140	0.059	0.845	0.007
25.0	0.075	0.759	1.170	0.044	0.869	0.013
30.0	0.072	0.863	1.202	0.036	0.894	0.020
35.0	0.062	0.954	1.236	0.035	0.920	0.026
40.0	0.040	1.025	1.271	0.050	0.941	0.030
45.0	0.015	1.086	1.302	0.082	0.964	0.033
50.0	-0.011	1.138	1.332	0.127	0.987	
55.0	-0.039	1.180	1.363	0.177	1.011	
60.0	-0.068	1.211	1.398	0.231	1.036	





TEST 3.1

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.625 metres
Transverse Centre of Gravity	0.000 metres
Equilibrium GM	0.326 metres
Equilibrium Heel Angle	0.000 degrees
Equilibrium Draught	1.144 metres
Equilibrium Trim Between Marks	0.117 metres by the stern
Angle of Vanishing Stability	37.7 degrees
Maximum GZ	0.085 metres
Maximum GZ Angle	18.7 degrees

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	0.000	0.000	1.085	0.117	0.783	0.000
5.0	0.028	0.170	1.089	0.112	0.787	0.001
6.2	0.035	0.210	1.091	0.109	0.789	0.002
10.0	0.055	0.338	1.100	0.099	0.800	0.005
12.2	0.067	0.410	1.106	0.090	0.809	0.007
15.0	0.079	0.499	1.116	0.079	0.821	0.011
20.0	0.084	0.640	1.139	0.060	0.845	0.018
25.0	0.072	0.759	1.169	0.046	0.869	0.025
30.0	0.050	0.863	1.202	0.037	0.894	0.030
35.0	0.021	0.954	1.236	0.036	0.920	0.034
40.0	-0.019	1.025	1.270	0.051	0.941	
45.0	-0.063	1.086	1.301	0.085	0.964	
50.0	-0.107	1.138	1.330	0.131	0.987	
55.0	-0.151	1.180	1.361	0.182	1.011	
60.0	-0.196	1.211	1.395	0.238	1.036	





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TEST 3.2

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.625 metres
Transverse Centre of Gravity	0.033 metres
Equilibrium GM	0.321 metres
Equilibrium Heel Angle	5.828 degrees to stbd
Equilibrium Draught	1.146 metres
Equilibrium Trim Between Marks	0.110 metres by the stern
Angle of Vanishing Stability	34.2 degrees to stbd 40.6 degrees to port
Maximum GZ	0.053 metres to stbd 0.116 metres to port
Maximum GZ Angle	19.3 degrees to stbd 18.5 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.033	0.000	1.085	0.117	0.783	0.002
5.0	-0.005	0.170	1.089	0.112	0.787	0.000
5.8	0.000	0.198	1.090	0.110	0.789	0.000
6.2	0.002	0.210	1.091	0.109	0.789	0.000
10.0	0.023	0.338	1.100	0.099	0.800	0.001
12.2	0.034	0.410	1.106	0.090	0.809	0.002
15.0	0.047	0.499	1.116	0.079	0.821	0.004
20.0	0.053	0.640	1.139	0.060	0.845	0.008
25.0	0.042	0.759	1.169	0.046	0.869	0.013
30.0	0.022	0.863	1.202	0.037	0.894	0.015
35.0	-0.006	0.954	1.236	0.036	0.920	
40.0	-0.045	1.025	1.270	0.051	0.941	
45.0	-0.086	1.086	1.301	0.085	0.964	
50.0	-0.128	1.138	1.330	0.131	0.987	
55.0	-0.170	1.180	1.361	0.182	1.011	
60.0	-0.213	1.211	1.395	0.238	1.036	





TEST 3.3

Displacement	14.417 tonnes
Longitudinal Centre of Gravity	-4.902 metres
Vertical Centre of Gravity	1.625 metres
Transverse Centre of Gravity	0.066 metres
Equilibrium GM	0.302 metres
Equilibrium Heel Angle	11.788 degrees to stbd
Equilibrium Draught	1.151 metres
Equilibrium Trim Between Marks	0.092 metres by the stern
Angle of Vanishing Stability	28.3 degrees to stbd 43.3 degrees to port
Maximum GZ	0.022 metres to stbd 0.147 metres to port
Maximum GZ Angle	19.6 degrees to stbd 18.3 degrees to port

Heel Anale	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.066	0.000	1.085	0.117	0.783	0.007
5.0	-0.038	0.170	1.089	0.112	0.787	0.002
6.2	-0.031	0.210	1.091	0.109	0.789	0.002
10.0	-0.010	0.338	1.100	0.099	0.800	0.000
11.8	0.000	0.397	1.105	0.092	0.807	0.000
12.2	0.002	0.410	1.106	0.090	0.809	0.000
15.0	0.015	0.499	1.116	0.079	0.821	0.000
20.0	0.022	0.640	1.139	0.060	0.845	0.002
25.0	0.012	0.759	1.169	0.046	0.869	0.004
30.0	-0.007	0.863	1.202	0.037	0.894	
35.0	-0.033	0.954	1.236	0.036	0.920	
40.0	-0.070	1.025	1.270	0.051	0.941	
45.0	-0.109	1.086	1.301	0.085	0.964	
50.0	-0.149	1.138	1.330	0.131	0.987	
55.0	-0.189	1.180	1.361	0.182	1.011	
60.0	-0.229	1.211	1.395	0.238	1.036	





TEST 4.1

Displacement	17.773 tonnes
Longitudinal Centre of Gravity	-5.220 metres
Vertical Centre of Gravity	1.616 metres
Transverse Centre of Gravity	0.000 metres
Equilibrium GM	0.275 metres
Equilibrium Heel Angle	0.000 degrees
Equilibrium Draught	1.259 metres
Equilibrium Trim Between Marks	0.415 metres by the stern
Angle of Vanishing Stability	26.5 degrees to stbd 26.5 degrees to port
Maximum GZ	0.044 metres to stbd 0.044 metres to port
Maximum GZ Angle	12.3 degrees to stbd 12.3 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
aegrees	metres	metres	metres	metres	metres	metres.rad
0.0	0.000	0.000	1.052	0.415	0.872	0.000
5.0	0.024	0.165	1.054	0.414	0.876	0.001
6.2	0.030	0.204	1.054	0.414	0.878	0.002
10.0	0.042	0.323	1.054	0.428	0.886	0.004
12.2	0.044	0.386	1.054	0.443	0.892	0.006
15.0	0.042	0.460	1.055	0.467	0.900	0.008
20.0	0.029	0.581	1.062	0.516	0.916	0.011
25.0	0.009	0.692	1.077	0.568	0.934	0.013
30.0	-0.023	0.785	1.091	0.641	0.950	
35.0	-0.063	0.864	1.111	0.720	0.964	
40.0	-0.106	0.932	1.131	0.807	0.978	
45.0	-0.149	0.993	1.150	0.902	0.993	
50.0	-0.191	1.047	1.170	1.003	1.009	
55.0	-0.233	1.091	1.192	1.105	1.025	
60.0	-0.273	1.126	1.219	1.203	1.042	





TEST 4.2

Displacement	17.773 tonnes
Longitudinal Centre of Gravity	-5.220 metres
Vertical Centre of Gravity	1.616 metres
Transverse Centre of Gravity	0.027 metres
Equilibrium GM	0.285 metres
Equilibrium Heel Angle	5.518 degrees to stbd
Equilibrium Draught	1.261 metres
Equilibrium Trim Between Marks	0.414 metres by the stern
Angle of Vanishing Stability	20.9 degrees to stbd 30.1 degrees to port
Maximum GZ	0.018 metres to stbd 0.070 metres to port
Maximum GZ Angle	12.5 degrees to stbd 12.2 degrees to port

Heel Anale	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve Area
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.027	0.000	1.052	0.415	0.872	0.001
5.0	-0.003	0.165	1.054	0.414	0.876	0.000
5.5	0.000	0.182	1.054	0.414	0.876	0.000
6.2	0.003	0.204	1.054	0.414	0.878	0.000
10.0	0.015	0.323	1.054	0.428	0.886	0.001
12.2	0.018	0.386	1.054	0.443	0.892	0.001
15.0	0.016	0.460	1.055	0.467	0.900	0.002
20.0	0.003	0.581	1.062	0.516	0.916	0.003
25.0	-0.016	0.692	1.077	0.568	0.934	
30.0	-0.046	0.785	1.091	0.641	0.950	
35.0	-0.085	0.864	1.111	0.720	0.964	
40.0	-0.127	0.932	1.131	0.807	0.978	
45.0	-0.168	0.993	1.150	0.902	0.993	
50.0	-0.209	1.047	1.170	1.003	1.009	
55.0	-0.248	1.091	1.192	1.105	1.025	
60.0	-0.287	1.126	1.219	1.203	1.042	





TEST 4.3

Displacement	17.773 tonnes
Longitudinal Centre of Gravity	-5.220 metres
Vertical Centre of Gravity	1.616 metres
Transverse Centre of Gravity	0.040 metres
Equilibrium GM	0.149 metres
Equilibrium Heel Angle	8.961 degrees to stbd
Equilibrium Draught	1.266 metres
Equilibrium Trim Between Marks	0.423 metres by the stern
Angle of Vanishing Stability	16.8 degrees to stbd 31.5 degrees to port
Maximum GZ	0.005 metres to stbd 0.083 metres to port
Maximum GZ Angle	12.5 degrees to stbd 12.2 degrees to port

Heel Angle	Righting GZ	Lever KN	Waterline	Trim	VCB	GZ Curve
degrees	metres	metres	metres	metres	metres	metres.rad
0.0	-0.040	0.000	1.052	0.415	0.872	0.003
5.0	-0.015	0.165	1.054	0.414	0.876	0.000
6.2	-0.010	0.204	1.054	0.414	0.878	0.000
9.0	0.000	0.292	1.055	0.423	0.884	0.000
10.0	0.003	0.323	1.054	0.428	0.886	0.000
12.2	0.005	0.386	1.054	0.443	0.892	0.000
15.0	0.003	0.460	1.055	0.467	0.900	0.000
20.0	-0.009	0.581	1.062	0.516	0.916	
25.0	-0.028	0.692	1.077	0.568	0.934	
30.0	-0.057	0.785	1.091	0.641	0.950	
35.0	-0.096	0.864	1.111	0.720	0.964	
40.0	-0.137	0.932	1.131	0.807	0.978	
45.0	-0.178	0.993	1.150	0.902	0.993	
50.0	-0.217	1.047	1.170	1.003	1.009	
55.0	-0.256	1.091	1.192	1.105	1.025	
60.0	-0.293	1.126	1.219	1.203	1.042	







Appendix 2

RINA Safety Committee review

FOR MARINE TECHNOLOGY AND INDUSTRIAL AERODYNAMICS

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To:	RINA Safety Committee	Date:	19/02/2024
From:	Matteo Scarponi, Wolfson Unit MTIA	Email:	wumtia@soton.ac.uk
Subject:	Wolfson Response to RINA Review	of Report	No 2900 Rev.10

1 INTRODUCTION

This document was prepared in response to a letter [1] by the RINA Safety Committee on Wolfson Unit report No. 2900 Rev. 10 'Evidencing a Seastate Allowance for Inshore Fishing Vessels by Model Scale Testing (Phase I)'. Report No. 2900 is referred to as 'Wolfson Report' in this document.

We wish to thank the RINA Safety Committee for convening an ad-hoc working group and for giving the Wolfson Unit the opportunity to respond to the Committee's queries.

Section 3 below presents our response to the RINA queries and Section 4 provides a proposed course of action.

2 BACKGROUND

The Wolfson report was funded by Research England through the Higher Education Innovation Fund (HEIF). The HEIF grant funding was awarded to the Wolfson Unit by the Southampton Marine and Maritime Institute (SMMI) following a competitive tendering process.

The project was led by Dr Matteo Scarponi PhD CEng MRINA, Principal Consulting Engineer in the Wolfson Unit. The National Federation of Fishermen's Organisations (NFFO) contributed specific industry expertise and the Marine Accident Investigation Branch (MAIB) provided a monohull design for testing. The Maritime and Coastguard Agency (MCA) acted as independent observers throughout the project. The tests, which were witnessed by NFFO and MCA representatives, were conducted and analysed by Wolfson Unit engineers.

The Wolfson Report was published on 12th December, 2023 following a consultation stage involving NFFO, MCA and MAIB.

3 RESPONSE TO RINA QUERIES

In the following we refer directly to specific sections of the RINA letter [1].



3.1 The report does not demonstrate that the flow of water aboard the vessel at experimental scale is an accurate simulation of the water behaviour at full scale.

The dead ship model tests described in the Wolfson Report are based on the Froude similarity, which ensures consistency between the model scale motions and the full scale motions. This not only applies to the hull motions in waves, but also to the motions of the entrained water.

Froude similarity requires geometric similarity and identical Froude number (ratio of inertia force to gravity force, or 'Fr') for the vessel and its scale model.

The physical testing of scale models in accordance with the Froude similarity is a recognised approach for assessing the damage stability characteristics of Ro-Ro passenger ships and other vessel types, and to generate realistic fluid flow within model scale cargo tanks for the measurement of sloshing loads on the tank walls. Formal testing protocols for these two ship types have been developed by the International Towing Tank Conference (ITTC) [2, 3].

To achieve Froude similarity and therefore accurate modelling of the water behaviour, we:

- a. chose a scale factor of 1:6 to minimise scale effects due to surface tension forces,
- b. built a geosim model (all length dimensions scaled by 6) to the appropriate specification, and
- c. ballasted the model to the appropriate test conditions (draught, LCG, GM, pitch and roll gyradii all scaled by 6) using standard Wolfson Unit procedures.

Further information on items a. to c. is provided below:

<u>a. Choice of scale</u> - If $Fr_{ship} = Fr_{model}$ then scale effects arise due to forces whose impact is greater at model scale than at full scale, such as viscosity and surface tension. Scale effects reduce with increasing model size, so can be minimised by selecting the appropriate scale in relation to the hull features and the scope of work.

A scale of 1:6 was selected for the Wolfson tests. Such a choice was affected by two conflicting requirements that are, minimise scale effects (model as large as possible) and produce the widest possible range of wave periods and wave heights within the wavemaker capability (model as small as possible).

More information on the similarity theory and scale effects is provided in [4] and the size of the openings of the model under test in relation to the ITTC recommendations [3] are detailed in Section 3.2 below.

<u>b. Model construction</u> - The model was constructed to standard Wolfson specification to ensure an accurate, stiff, dimensionally stable, and watertight hull. It was designated as M1203.

On the decked model variant, the deck camber, sheer, bulwark height and longitudinal position of the freeing ports mirrored the original design. A clear open deck was modelled, to avoid making arbitrary assumptions on deck fittings and fishing gear. On the open boat variant, the deck drainage system and bilge pump arrangement were agreed in discussion with NFFO and the MCA to represent a typical installation advised by the regulators for low-freeboard boats similar to the vessel tested.

<u>c. Model ballasting for Centre of Gravity (CG) position and inertia</u> - The chosen scale ensured compliance with standard model preparation procedures described in [5], such as:

- accurate ballasting by inclining in air, to achieve the desired displacement and CG position over the range of loading conditions tested;
- accurate modelling of longitudinal inertia, to achieve a pitch gyradius of 25% overall length;
- accurate modelling of transverse inertia, to achieve a roll gyradius of 33% overall beam.

3.2 Members of the group questioned whether the scupper openings and the downflooding ducts on the open boats accurately scaled behaviour from model to full size.

Physical model testing is often used in support of marine accident investigations into the progressive flooding, foundering and capsizing of marine craft. Reference [6] describes nine accident investigations worldwide that entailed model testing, six of which involved commercial fishing vessels. The Wolfson Unit conducted model scale tests into capsize and progressive flooding events [7, 8, 9, 10] as part of MAIB investigations and in aid of MCA research.

ITTC procedure [3] sets a best practice for conducting damage stability model experiments for design evaluation purposes and is based on a review of model tests on various ship types, including Ro-Ro passenger ships. These vessels present large open decks and are therefore vulnerable to capsizing in a damage state, due to large free surface effects. Crucially, procedure [3] covers 'modelling the transition from an intact hull to a damaged hull' at zero forward ship speed and aspects of it are therefore deemed relevant to the Wolfson Report.

In particular, [3] recommends a minimum scale ratio of 1:40 for Ro-Ro ships and a minimum model length of 3.0m, to ensure an accurate representation of the full scale vessel and minimise viscous effects due to flow through small openings. To that end, [3] recommends a minimum cross sectional area of 500mm² for cross-flooding arrangements and small diameter ducts.

The Wolfson model was built at a scale of 1:6 corresponding to 1.73m overall length. The resulting freeing port areas and the cross section of the deck drains are shown in Table 1 below. The radii of the model scale freeing port edges and deck drains were sharp to minimise surface tension effects.

In conclusion, the model scale areas of freeing ports and downflooding ducts significantly exceeded the ITTC requirement aimed at minimising viscous scale effects, and the model scale radii of the openings were sharp to minimise surface tension effects. This ensures minimisation of potential scale effects to robustly model the scaled behaviour.

Opening type	Model scale area (mm ²)	ITTC recommended area (mm ²)	Margin over ITTC recommended area [3]
Freeing port #1	1283	500	2.6x
Freeing port #2	1313	500	2.6x
Freeing port #3	2000	500	4.0x
Freeing port #4	1283	500	2.6x
Cross section of downflooding duct	2940	500	5.9x

Table 1 Size of model scale freeing ports and deck drains for fishing boat model M1203

3.3 The Figures 7 a, b and c and Figures 8 a, b, and c purport to show 'capsize boundaries', however almost all show no such thing as a boundary between conditions that could result in a capsize and those that do not.

The derivation of capsize boundaries in regular waves requires systematic testing over the widest possible range of model scale wave conditions permitted by the wavemaker, including combinations of wave height and period representing breaking waves and open ocean conditions. This, however, is outside the scope of the Wolfson Report.

The Wolfson Report uses the term 'capsize boundary' for consistency with Figures 30-38 of [11], where the tested combinations of regular wave height and wave period were plotted, capsize events highlighted and capsize trends identified as appropriate.

Figures 7 a-c and 8 a-c are intended to summarise all the runs performed and their outcome, and enable the reader to:

- a. assess the relative survivability of the decked and open configurations at the load and wave conditions tested, highlighting those that resulted in a capsize;
- b. compare the wave heights in which the decked model capsized to the <u>maximum recommended</u> wave height based on the residual freeboard, as advised in the Wolfson Notice;
- c. populate Figure 11 of the Wolfson report (see Figure 1 of this document) to enable validation of the predicted critical wave height based on the residual stability. This area is discussed further in section 3.6 of this document.

The maximum recommended wave height advised in the Stability Notice is based on the top of the red zone freeboard (210mm for the design tested) and any further freeboard reduction will reduce that. Capsizes can occur below the maximum recommended wave height, as ID 4.3 did, because of the approximate nature of the guidance and range of freeboards in each zone. Adequate information on the approximate nature of the guidance is part of the fishers' stability awareness syllabi and training.

3.4 Only figure 8b has such a boundary (denoted by the cases where capsize occurs), but even this is inconclusive. For this loading condition experiments need to be conducted at smaller wave heights for periods of 3, 4 and 7 seconds to demonstrate that the observed capsize is at, or close to, the boundary.

Figure 8b refers to configuration ID 4.2 which, as pointed out in the Wolfson Report, is an overloaded and top heavy loading condition with 5.5° equilibrium heel that would fail all the formal stability criteria for new vessels. The full scale decked vessel would have 20mm negative residual freeboard at that condition (deck edge underwater) and the low-side Wolfson Mark would probably be immersed regardless of its longitudinal position.

At 15° residual range, a maximum righting moment of just 300kg.m and a negative residual freeboard, there is no doubt that the vessel is vulnerable to capsizing in any seastate. Testing decked configurations with no residual stability such as ID 4.2 did not aim at identifying a safe boundary of operation at these unsafe conditions, and readers should not infer from Figure 8b or the Wolfson Notice that it is acceptable to operate in some seastates with no residual freeboard.

Testing at ID 4.2 is deemed conclusive because it fulfils the three key objectives stated in Section 3.3 above.

<u>Small</u> wave height reductions at 3, 4 and 7s period may have resulted in additional capsize data but no new significant information on the vessel's motions in waves or capsize mechanism, hence such additional runs were not performed. <u>Large</u> wave height reductions at each period would have resulted in shallower waves and therefore less severe conditions, as the decked vessel capsize mechanism was driven by wave steepness. Again, we did not see any merit in performing these additional runs.

3.5 In addition Figure 6 indicates that for several loading condition no experiments have been conducted at longer waver periods, despite the capsizes observed for loading condition 4.2 at wave periods of 6 and 7 seconds.

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In the Wolfson report, para. 8b of the decked vessel discussion states 'The capsize mechanism entailed rolling of the model to large windward angles, causing immersion of the windward freeing ports and resulting in the accumulation of water on deck'. In waves, a vessel is forced to roll by the wave slope on which it floats [12]. Observation of the decked model behaviour in waves confirmed that, given the loading condition, the roll motions increased with wave steepness, thus increasing the likelihood of a capsize.

Testing at longer wave periods whilst maintaining wave steepness requires increasing wave height. This may add potential capsize data but also has these consequences:

- add capsize points well above the <u>predicted critical</u> line of Figure 2, thus devaluing the validation exercise;
- risk of exceeding the wavemaker envelope, causing damage to the facility
- reduction of the run rate: as larger waves carry more energy, they take longer to dissipate at the end of a run, thus increasing downtime between subsequent runs and reducing efficiency.

Therefore the bulk of the tests focused on the left hand side region of Figure 6 ie low wave heights combined with short periods up to and including 5 seconds.



3.6 In the Executive Summary it is stated that the physical testing 'confirms the merit of a seastate based allowance, since all capsize events occurred at combinations of stability and wave height deemed unsafe by the Wolfson Method.' This statement cannot be supported by the group without further tests being undertaken that demonstrate that with wave heights that are deemed safe by the Wolfson Method capsizes do not occur.

In our view the statement quoted above is sufficiently evidenced in Figure 1 below, where at least one 'no capsize' run underpins all 'capsize' runs at the three capsize condition tested that are, 2.3, 4.2 and 4.3. In particular, the decked model survives at Runs 33, 24 and 17 but capsizes at Runs 35, 25 and 14.

These results are in keeping with other 'stress testing' exercises conducted by the Wolfson Unit, such as that described in [13] and summarised in Figure 2.













4 PROPOSED COURSE OF ACTION FOLLOWING RINA REVIEW

- a. Following on from queries 3.3 and 3.4 above to replace the term 'capsize boundary' with 'tests performed' in the captions of Figure 7a to c and 8a to c.
- b. Following on from query 3.3 above to remove the references to a 'maximum recommended wave height' from Figures 8b and 8c to avoid the impression that it may be safe to operate with negative freeboards.
- c. Following on from query 3.3 above add the text 'No residual freeboard = no maximum recommended wave height' to the captions of Figures 8b and 8c.
- d. Reword the first sentence of Section 8 (decked vessel discussion) Item e. as follows: 'The model capsized below the maximum recommended wave height based on the residual freeboard, but well above the predicted critical height based on the residual stability.'
- e. To include this document in the Wolfson Report as 'Appendix 2 Discussion'. The Wolfson Report, edited as above, will then be published as Rev.11 which will replace and supersede Rev.10

5 REFERENCES

- [1] Watts, S. Letter from RINA Safety Committee to The Seafarers' Charity, January 2024. Confidential communication.
- [2] International Towing Tank Conference, '<u>Sloshing Model Tests</u>', ITTC Recommended Procedures and Guidelines, 7.5.-02-07-02.7 Rev.1, 2021. Last accessed February 2024.
- [3] International Towing Tank Conference, '<u>Model Tests on Damage Stability in Waves</u>', ITTC Recommended Procedures and Guidelines, 7.5.-02-07-04.2 Rev.2, 2014. Last accessed February 2024.
- [4] Heller, V. 'Development of Wave Devices from Initial Conception to Commercial Demonstration', in 'Comprehensive Renewable Energy', Elsevier, 2012.
- [5] Claughton, A. R., 'Tank Testing', Wolfson Marine Craft Unit Technical Note No. 20, February 1979.
- [6] Pérez-Rojas, L. et al. '<u>Review of the ship accidents investigations presented at the STAB</u> <u>Workshops/Conferences</u>', Proc. of the 10th International Ship Stability Workshop. Last accessed February 2024.
- [7] Wolfson Unit, 'Model tests to simulate the sinking of a fishing vessel'. Wolfson Unit report 1116 for MAIB, March 1993.
- [8] Wolfson Unit, 'Further model tests to simulate the sinking of a fishing vessel'. Wolfson Unit report 1126 for MAIB, July 1993.
- [9] Wolfson Unit, 'MCA Research Project 433: The deployment of liferafts carried on UK registered fishing vessels'. Report 1442/1, Maritime & Coastguard Agency, UK.
- [10] Wolfson Unit, '<u>MFV Amber Rose Model capsizing tests</u>'. Wolfson Unit report 1523 in support of MAIB investigation 24/2000, May 2000.
- [11] Wolfson Unit, '<u>MCA Research Project 509: High Speed Craft Evaluation of Existing Criteria, Final</u> <u>Report</u>'. Report No.1779, Maritime & Coastguard Agency, UK.
- [12] Wolfson Unit, 'MCA Research Project 529: Loading Guidance for FVs Less than 12m Registered Length, Phase I'. Report No.1778, Maritime & Coastguard Agency, UK.
- [13] Scarponi, M., 'Stress Tests on the Wolfson Stability Method for Small Fishing Vessels', Lloyd's Register Foundation Small Grant Final Report ref. SG4/100045, March 2022. Last accessed February 2024.

Version	Date issued	Change log
2900_Rev10	Dec 2023	Version reviewed by RINA SC
2900_Rev11	Oct 2024	 Section 4 items a, b, c, d, e – all proposed changes applied 'Similar' vessels now defined in footnote to Section 11 Error bars and run nos. now added to selected data points of Figure 11 Figure 20 appended and context given in Section 8 Paragraph f

List of changes following RINA Safety Committee (SC) review

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